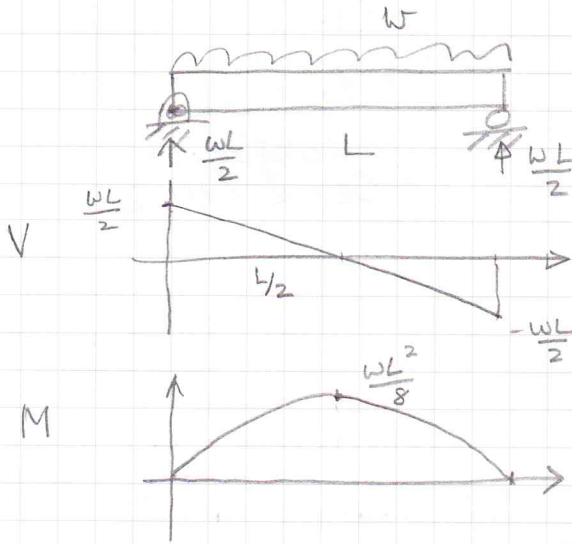


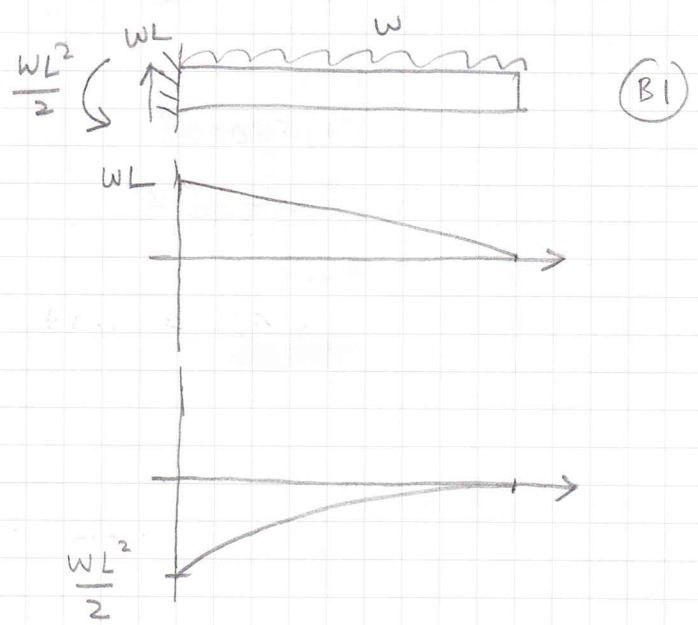
Date .

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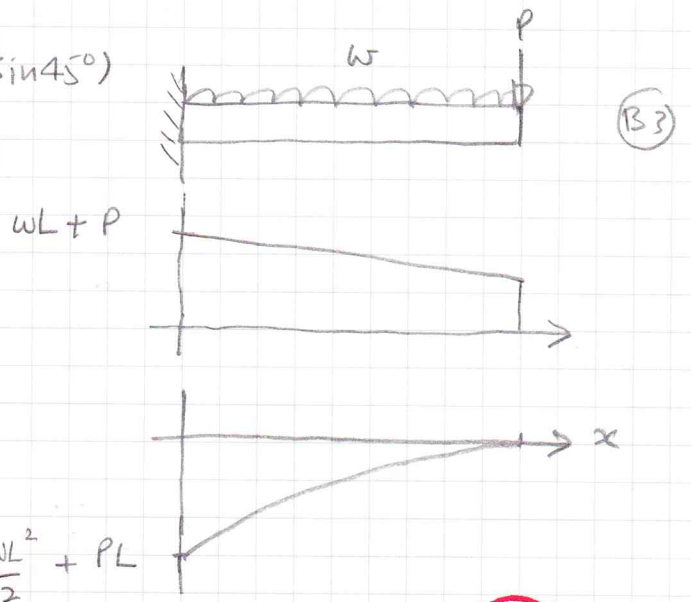
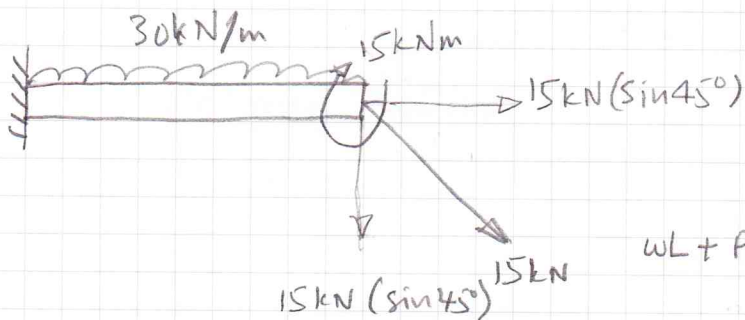
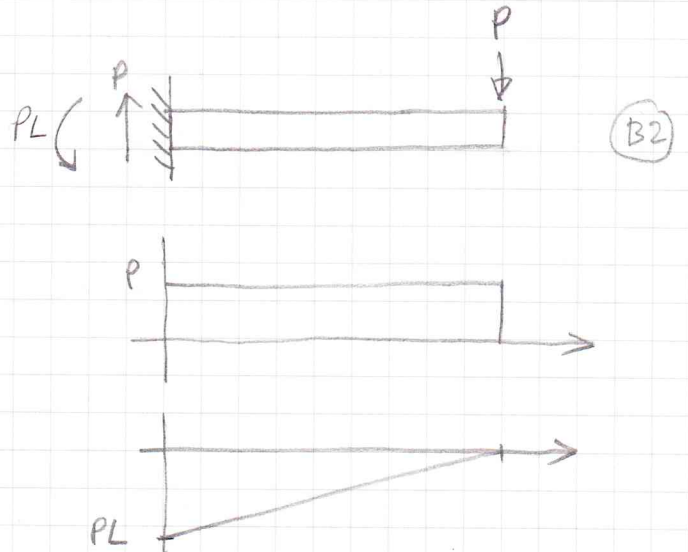
(A) Simply Supported beam



(B) cantilevered beam



- Given
- $w = 30 \text{ kN/m}$
 - $T = 15 \text{ kNm}$
 - $L = 3 \text{ m}$
 - $\rho = 2.79 \text{ Mg/m}^3$
 - $E = 73.1 \text{ GPa}$
 - $\sigma_y = 414 \text{ MPa}$
 - $\phi = 150 \text{ mm}$



Define

- 1) External loading
- 2) Body force
- 3) reaction
- 4) internal loading

10

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The beam is under bending (1), axial (2), torsional (3)

and transverse shear (4) loading

① Bending loading.

Using BS diagram, maximum shear force and maximum moment can be calculated.

$$M_{\max} = \frac{wL^2}{2} + PL$$

$$= \frac{(30 \times 10^3)(3)^2}{2} + (15 \times 10^3)(\sin 45^\circ)(3)$$

$$= 135 \times 10^3 + 31.82 \times 10^3$$

$$= 166.82 \times 10^3$$

$$= 166.82 \text{ kNm}$$

$$V_{\max} = wL + P$$

$$= (30 \times 10^3)(3) + (15 \times 10^3)(\sin 45^\circ)$$

$$= 90 \times 10^3 + 10.61 \times 10^3$$

$$= 100.61 \times 10^3$$

$$= 100.61 \text{ kN}$$

(10)

Bending stress, $\sigma_b = \frac{M_{\max} \cdot c}{I}$

$$I = \frac{\pi}{64} \phi^4$$

$$= \frac{\pi}{64} (0.15)^4$$

$$= 24.85 \times 10^{-6} \text{ m}^4$$

Bending stress

$$\sigma_{\max} = \frac{166.82(10^3)(0.075)}{24.85 \times 10^{-6}}$$

$$= 503.48 \text{ MPa}$$

$$c = 0.15/2$$

$$= 0.075 \text{ m}$$

(5) ~~(10)~~

$$\begin{aligned}
 \textcircled{2} \text{ Axial loading, } \sigma_a &= P/A \\
 &= \frac{10.61 \times 10^3}{\frac{\pi}{4} (0.15)^2} \\
 &= 600.4 \text{ kN/m}^2 \quad \textcircled{5}
 \end{aligned}$$

$$\textcircled{3} \text{ Torsional loading, } \tau_{\max} = \frac{T r}{J}$$

$$\begin{aligned}
 J &= \frac{\pi \phi^4}{32} \\
 &= \frac{\pi (0.15)^4}{32} \\
 &= 49.7 \times 10^{-6} \text{ m}^4
 \end{aligned}$$

$$\begin{aligned}
 \tau_{\max} &= \frac{T r}{J} = \\
 &= \frac{(15 \times 10^3) (0.075)}{49.7 \times 10^{-6}} \\
 &= 22.64 \text{ MPa} \quad \textcircled{15}
 \end{aligned}$$

④ Transverse shear

$$\begin{aligned}
 \tau_v &= 1.5 \frac{V}{A} \\
 &= \frac{1.5 \times 100.61 \times 10^3}{\frac{\pi}{4} (0.15)^2} \\
 &= 8.54 \text{ MPa} \quad \textcircled{5}
 \end{aligned}$$

$$\begin{aligned}
 \text{Total stresses, } \sigma_T &= \sigma_a \pm \sigma_b + \tau_{\max} + \tau_v \\
 &= 0.6004 \pm 503.48 + 22.64 + 8.5 \\
 &= 535.26 \text{ MPa} \quad \text{tension} \\
 \text{and } &-471.76 \text{ MPa} \quad \text{compression}
 \end{aligned}$$

Therefore the beam is anticipating failure since the stresses are more than the yield stress ①5

$$\text{New } I = 61.2 \times 10^6 \text{ mm}^4 \quad \text{or} \quad 61.2 \times 10^{-6} \text{ m}^4$$

$$\text{Bending stress} = \frac{166.82 (10^3) (0.075)}{61.2 \times 10^{-6}} \quad \begin{array}{l} \text{assume } c \\ \text{is similar} \end{array}$$
$$= 204.44 \text{ MPa}$$

$$\text{Total stress} = 236.22 \text{ MPa} \quad \text{Tension}$$
$$\text{and} \quad - 172.44 \text{ MPa} \quad \text{Compression}$$

The beam is safe since both stresses are below the yield stress.

(15)